

## Die Storage Rack Analysis

Cross-members: 6"x6"x1/4" HSS tubing

Long. members: 6"x6"x1/4" HSS tubing  
W6"x20#

Fixed end reaction for the cross members (dist. load w lbs./in.)

$$FEA = w \cdot L / 2$$

Fixed end moment for the cross members (dist. load w lbs./in.)

$$FEM = w \cdot L^2 / 12 \quad \text{beam fixed at ends}$$

HSS tubing:  $s_y = 46 \text{ ksi}$

$$L = 96 \text{ in.}$$

$$w = 250 \text{ lbs./in}$$

$$FEA = 12,000 \text{ lbs.}$$

$$FEM = 192,000 \text{ in-lbs.}$$

$$A = 5.24 \text{ in}^2$$

$$I_x = 28.6 \text{ in}^4$$

$$S = 9.54 \text{ in}^3$$

$$s_b = 20,126 \text{ psi} \quad \text{calculated bending stress at top}$$

$$s_a = 30,360 \text{ psi} \quad \text{allowable bending stress}$$

### Shear Stress:

$$Q = 2.91 \text{ in}^3$$

$$t = 5,236 \text{ psi} \quad \text{calculated max. shear stress in side wall}$$

$$Q = 2.02 \text{ in}^3$$

$$t = 3,630 \text{ psi} \quad \text{calculated max. shear stress at corner}$$

$$s_b = 18,576 \text{ psi} \quad \text{calculated bending stress at corner}$$

### Combined Stress:

$$s = 19,260 \text{ psi} \quad \text{calculated stress at corner}$$

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Long. members: 6"x6"x1/4" HSS tubing  
W6"x20#

I-beam:  $s_y = 36 \text{ ksi}$

$L = 144 \text{ in.}$

$A = 5.87 \text{ in}^2$   
 $I_x = 41.4 \text{ in}^4$   
 $S = 13.4 \text{ in}^3$

### Beam Forces and Moments:

$P = 12,000 \text{ lbs.}$  from loading on cross-member  
 $a = 35 \text{ in.}$   
 $b = 109 \text{ in.}$

$R_1 = 10,218 \text{ lbs.}$   
 $R_2 = 1,782 \text{ lbs.}$

### Bending moment from concentrated load:

$M_1 = -240,645 \text{ in-lbs.}$  @ end of beam  
 $M_a = 116,980 \text{ in-lbs.}$  @ intersection of cross-member and beam  
 $M_c = 51,042 \text{ in-lbs.}$  @ center of beam  
 $M_b = -14,897 \text{ in-lbs.}$  @ intersection of cross-member and beam  
 $M_2 = -77,271 \text{ in-lbs.}$  @ end of beam

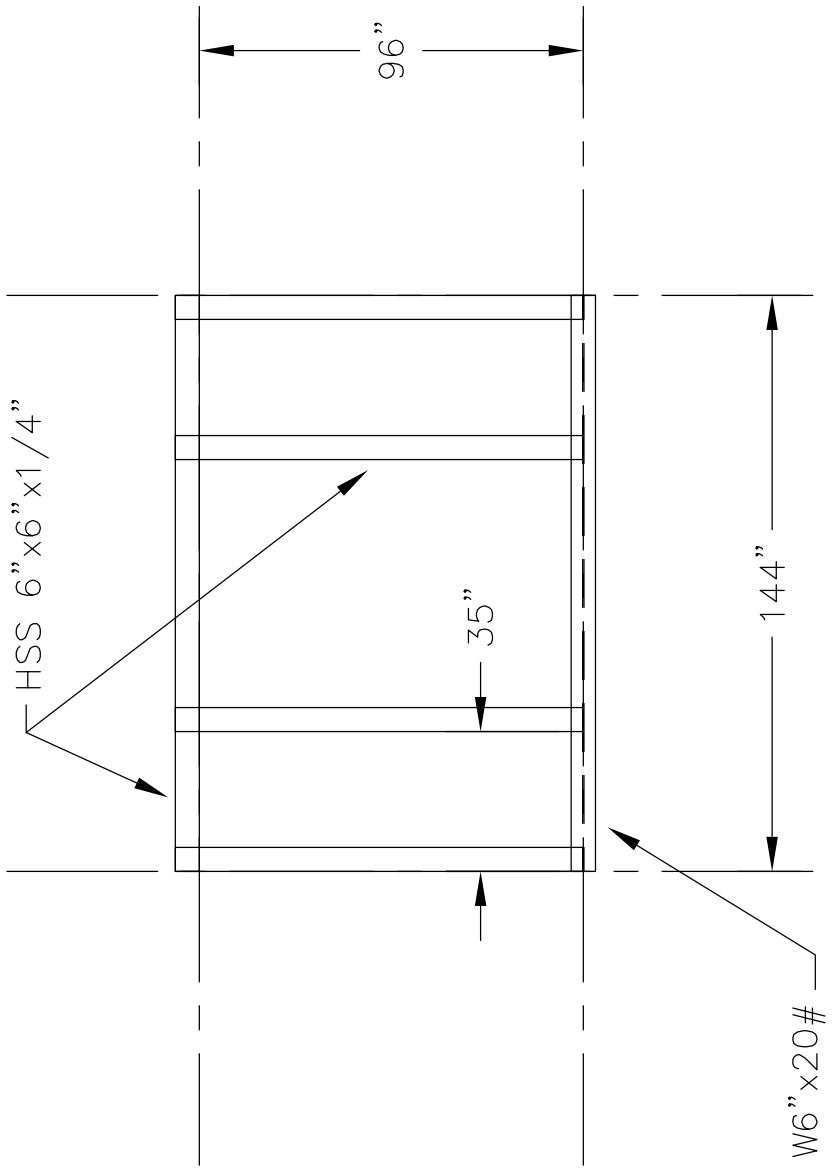
### Bending moment from distributed load:

$w = 0 \text{ lbs/in.}$   
 $M_1 = 0 \text{ in-lbs.}$  @ end of beam  
 $M_a = 0 \text{ in-lbs.}$  @ intersection of cross-member and beam  
 $M_c = 0 \text{ in-lbs.}$  @ center of beam  
 $M_b = 0 \text{ in-lbs.}$  @ intersection of cross-member and beam  
 $M_2 = 0 \text{ in-lbs.}$  @ end of beam

### Combined Bending Moment:

$M_1 = -317,917 \text{ in-lbs.}$  @ end of beam  
 $M_a = 102,083 \text{ in-lbs.}$  @ intersection of cross-member and beam  
 $M_c = 102,083 \text{ in-lbs.}$  @ center of beam  
 $M_b = 102,083 \text{ in-lbs.}$  @ intersection of cross-member and beam  
 $M_2 = -317,917 \text{ in-lbs.}$  @ end of beam

$s_b = 23,725 \text{ psi}$  calculated bending stress at top flange  
 $s_a = 23,760 \text{ psi}$  allowable bending stress

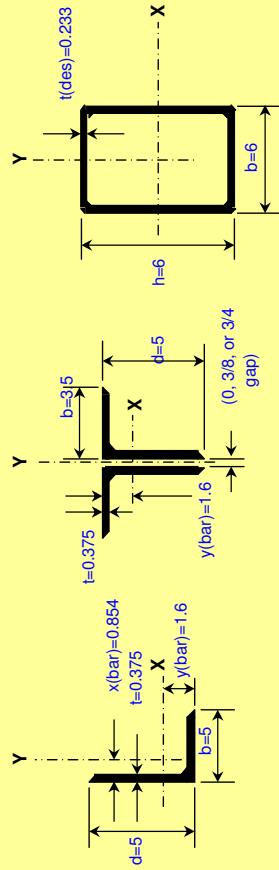


AISC 13th EDITION MEMBER DIMENSIONS AND PROPERTIES VIEWER

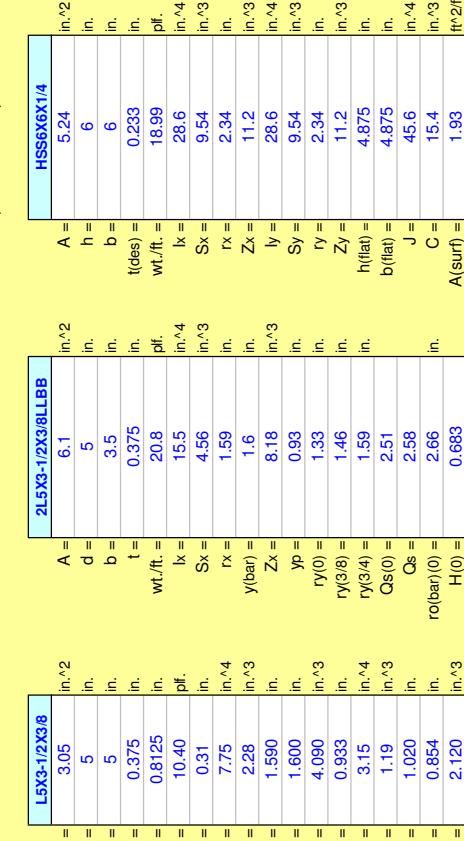
W, S, M, HP Shapes

C, MC Shapes

WT, ST, MT Shapes



W6X20		C10X5.3		WT5X11	
A =	5.87	in.^2	A =	3.24	in.^2
d =	6.2	in.	d =	5.090	in.
w =	0.26	in.	tw =	0.240	in.
f =	6.02	in.	bf =	5.750	in.
b =			tf =	0.360	in.
t =			k(des) =	0.6600	in.
ff =			k(det) =	0.9375	in.
bf =			gage =	2.75	in.
tt =			wt/ft =	11.00	plf.
ff =			bi/(2^t ft) =	7.990	plf.
k =	0.365	in.	d/tw =	21.200	in.^2
k =	0.615	in.	Ix =	6.88	in.
T =	0.875	in.	Sx =	1.72	in.
gage =	0.5625	in.	Rx =	1.46	in.
ts =	4.5	in.	y(bar) =	1.070	in.
ho =	3.5	in.	Zx =	3.02	in.^2
wt/ft =	20	plf.	yp =	0.282	in.
(ff) =	8.25	in.	ly =	5.71	in.^2
y =	18.7	in.^4	Sy =	1.99	in.
x =	41.4	in.^4	Ry =	1.33	in.
x =	13.4	in.^3	Zy =	3.05	in.^4
x =	2.66	in.	Qs(50) =	0.837	in.
x =	15	in.^3	J =	0.112	in.^4
y =	13.3	in.^4	Cw =	0.107	in.
y =	4.41	in.^3	a =	1.53	in.
x =	1.5	in.	ro(bar) =	2.16	in.
x(bar) =	6.72	in.^3			
Zy =					
xp =					
J =					
Cw =					
a =					
ro(bar) =					



**Round HSS & Pipes**

H(3/8) =	0.718
ro(3/4) =	1
H(3/4) =	0.983

in. in.

O.D.=8.625      t(nom)=0.3

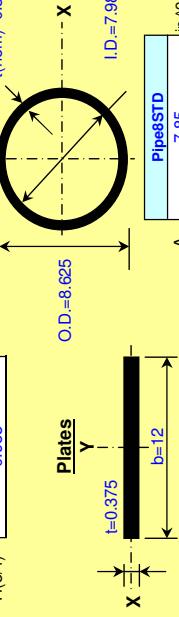
I.D.=7.96

**Plates**

t=0.375
b=12
b=12

in. in. in.

PipeSTD  
7.96



$t =$	<b>0.375</b>	O.D. =	8.625
$b =$	<b>12</b>	I.D. =	7.981
$wt./ft. =$	<b>15.31</b>	$t(nom) =$	0.322
$A =$	<b>4.500</b>	$t(des) =$	0.3
$I_x =$	<b>0.053</b>	$wt./ft =$	28.60
$S_x =$	<b>0.281</b>	$I_x = ly =$	68.1
$r_x =$	<b>0.108</b>	$S_x = Sy =$	15.8
$I_y =$	<b>54.000</b>	$r_x = ly =$	2.95
$S_y =$	<b>9.000</b>	$Z_x = Zy =$	20.8
$r_y =$	<b>3.464</b>	$I_y =$	136
$J =$	<b>54.053</b>	$C =$	---

**Reference:** The shapes contained in this database are taken from the AISC Version 13 "Shapes Database" CD-ROM Version (12/2005), as well as those listed in the AISC 13th Edition Manual of Steel Construction (12/2005).

**NOMENCLATURE FOR AISC VERSION 13.0 MEMBER PROPERTIES AND DIMENSIONS:**

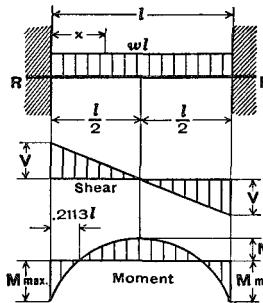
A = Cross-sectional area of member (in.^2)	d = Depth of member, parallel to Y-axis (in.)
h = Depth of member, parallel to Y-axis (in.)	tw = Thickness of web of member (in.)
bt = Width of flange of member, parallel to X-axis (in.)	b = Width of member, parallel to X-axis (in.)
tr = Thickness of flange of member (in.)	tr = Thickness of flange of member (in.)
k1 = Distance from outer face of flange to web toe of fillet (in.)	k = Distance between fillets for wide-flange or channel shape = d(nom)-2^*k(cet) (in.)
k2 = Distance from web centerline to flange toe of fillet (in.)	T = Standard gage (bolt spacing) for member (in.) ( <i>Note: gages for angles are available by viewing comment box at cell K18.</i> )
Sx = Elastic section modulus of member taken about X-axis (in.^3)	Iy = Moment of inertia of member taken about X-axis (in.^4)
Sy = Elastic section modulus of member taken about Y-axis (in.^3)	Iy = Moment of inertia of member taken about Y-axis (in.^4)
Zx = Plastic section modulus of member taken about X-axis (in.^3)	Iy = Radius of gyration of member taken about Y-axis (in.^3)
Zy = Plastic section modulus of member taken about Y-axis (in.^3)	rs = $\text{SQRT}(\text{Iy}/\text{Cw})/\text{Sx}$ (in.)
rs = $\text{SQRT}(\text{Iy}/\text{Cw})/\text{Sx}$ (in.)	x0 = horizontal distance from designated member edge to plastic neutral axis (in.)
x0 = horizontal distance from designated member edge to plastic neutral axis (in.)	yp = vertical distance from designated member edge to plastic neutral axis (in.)
ho = Distance between centroid of flanges, d-tr (in.)	ho = Distance between centroid of flanges, d-tr (in.)
J = Torsional moment of inertia of member (in.^4)	J = Torsional moment of inertia of member (in.^4)
Cw = Warping constant (in.^6)	C = Torsional constant for HSS shapes (in.^3)
C = Torsional constant for HSS shapes (in.^3)	a = $\text{SQRT}(\text{E}^*\text{Cw}^*\text{G}^*\text{J})$ (in.)
a = Torsional property, a = $\text{SQRT}(\text{E}^*\text{Cw}^*\text{G}^*\text{J})$ (in.)	E = Modulus of steel = 29,000 ksi
E = Modulus of steel = 29,000 ksi	G = Shear modulus of elasticity of steel = 11,200 ksi
G = Shear modulus of elasticity of steel = 11,200 ksi	W <sub>ho</sub> = Normalized warping function at a point at the flange edge (in.^2)
W <sub>ho</sub> = Normalized warping function at a point at the flange edge (in.^2)	Sw = Warping statical moment at a point on the cross section (in.^4)
Sw = Warping statical moment at a point on the cross section (in.^4)	Q <sub>r</sub> = Statical moment for a point in the flange directly above the vertical edge of the web (in.^3)
Q <sub>r</sub> = Statical moment for a point in the flange directly above the vertical edge of the web (in.^3)	Q <sub>w</sub> = Statical moment at the mid-depth of the section (in.^3)
Q <sub>w</sub> = Statical moment at the mid-depth of the section (in.^3)	x <sub>(bar)</sub> = Distance from outside face of web of channel shape or outside face of angle leg to Y-axis (in.)
x <sub>(bar)</sub> = Distance from outside face of flange of WT or angle leg to Y-axis (in.)	y <sub>(bar)</sub> = Horizontal distance from the outer edge of a channel web to its shear center (in.) = (approx.) $\text{tr}^*(\text{d}-\text{tr})^*/2^*(\text{b}-\text{tw})^*/2^*(4^*\text{k}_x-\text{tw})/2$
y <sub>(bar)</sub> = Horizontal distance from the outer edge of a channel web to its shear center (in.) = (approx.) $\text{tr}^*(\text{d}-\text{tr})^*/2^*(\text{b}-\text{tw})^*/2^*(4^*\text{k}_x-\text{tw})/2$	x <sub>o</sub> = x-coordinate of shear center with respect to the centroid of the section (in.)
x <sub>o</sub> = x-coordinate of shear center with respect to the centroid of the section (in.)	y <sub>o</sub> = y-coordinate of shear center with respect to the centroid of the section (in.)
y <sub>o</sub> = y-coordinate of shear center with respect to the centroid of the section (in.)	yo = Polar radius of gyration about the shear center = $\text{SQRT}(\text{x}_o^{\text{a}}+2\text{y}_o^{\text{a}}+2^*(\text{k}_x+\text{k}_y)/\text{A})$ (in.)
yo = Polar radius of gyration about the shear center = $\text{SQRT}(\text{x}_o^{\text{a}}+2\text{y}_o^{\text{a}}+2^*(\text{k}_x+\text{k}_y)/\text{A})$ (in.)	H = Flexural constant, $H = 1^*(\text{x}_o^{\text{a}}/2+\text{y}_o^{\text{a}}/2)/\text{t}(\text{b}-\text{tr})^*/2$
H = Flexural constant, $H = 1^*(\text{x}_o^{\text{a}}/2+\text{y}_o^{\text{a}}/2)/\text{t}(\text{b}-\text{tr})^*/2$	LLBB = Long legs back-to-back for double angles
LLBB = Long legs back-to-back for double angles	SLBB = Short legs back-to-back for double angles
SLBB = Short legs back-to-back for double angles	h <sub>(flat)</sub> = The workable flat (straight) dimension along the height, h (in.)
h <sub>(flat)</sub> = The workable flat (straight) dimension along the width, b (in.)	h <sub>(flat)</sub> = The workable flat (straight) dimension along the height, h (in.)
h <sub>(flat)</sub> = The workable flat (straight) dimension along the width, b (in.)	A <sub>(surf)</sub> = The total surface area of a rectangular or square HSS section (ft.^2/ft.)
A <sub>(surf)</sub> = The total surface area of a rectangular or square HSS section (ft.^2/ft.)	STD = Standard weight (Schedule 40) pipe section
STD = Standard weight (Schedule 40) pipe section	XS = Extra strong (Schedule 80) pipe section
XS = Extra strong (Schedule 80) pipe section	XXS = Double-extra strong pipe section

# BEAM DIAGRAMS AND FORMULAS

## For various static loading conditions

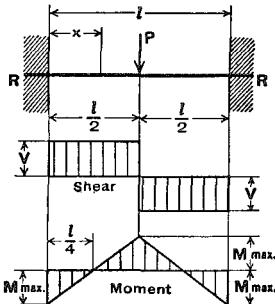
For meaning of symbols, see page 2 - 293

### 15. BEAM FIXED AT BOTH ENDS—UNIFORMLY DISTRIBUTED LOADS



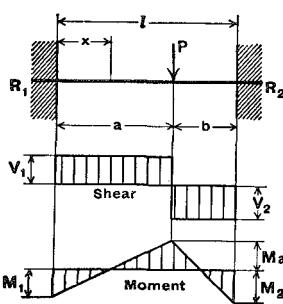
Total Equiv. Uniform Load . . . . .	$\frac{2wl}{3}$
$R = V$ . . . . .	$\frac{wl}{2}$
$V_x$ . . . . .	$w\left(\frac{l}{2} - x\right)$
M max. (at ends) . . . . .	$\frac{wl^2}{12}$
$M_1$ (at center) . . . . .	$\frac{wl^2}{24}$
$M_x$ . . . . .	$\frac{w}{12}(6lx - l^2 - 6x^2)$
$\Delta_{\max.}$ (at center) . . . . .	$\frac{wl^4}{384EI}$
$\Delta_x$ . . . . .	$\frac{wx^2}{24EI}(l - x)^2$

### 16. BEAM FIXED AT BOTH ENDS—CONCENTRATED LOAD AT CENTER



Total Equiv. Uniform Load . . . . .	$P$
$R = V$ . . . . .	$\frac{P}{2}$
M max. (at center and ends) . . . . .	$\frac{Pl}{8}$
$M_x$ (when $x < \frac{l}{2}$ ) . . . . .	$\frac{P}{8}(4x - l)$
$\Delta_{\max.}$ (at center) . . . . .	$\frac{Pl^3}{192EI}$
$\Delta_x$ (when $x < \frac{l}{2}$ ) . . . . .	$\frac{Px^2}{48EI}(3l - 4x)$

### 17. BEAM FIXED AT BOTH ENDS—CONCENTRATED LOAD AT ANY POINT



$R_1 = V_1$ (max. when $a < b$ ) . . . . .	$\frac{Pb^2}{l^3}(3a + b)$
$R_2 = V_2$ (max. when $a > b$ ) . . . . .	$\frac{Pa^2}{l^3}(a + 3b)$
$M_1$ (max. when $a < b$ ) . . . . .	$\frac{Pab^2}{l^2}$
$M_2$ (max. when $a > b$ ) . . . . .	$\frac{Pa^2b}{l^2}$
$M_a$ (at point of load) . . . . .	$\frac{2Pa^2b^2}{l^3}$
$M_x$ (when $x < a$ ) . . . . .	$R_1x - \frac{Pab^2}{l^2}$
$\Delta_{\max.}$ (when $a > b$ at $x = \frac{2al}{3a+b}$ ) . . . . .	$\frac{2Pa^2b^2}{3EI(3a+b)^2}$
$\Delta_a$ (at point of load) . . . . .	$\frac{Pa^3b^3}{3EI/l^3}$
$\Delta_x$ (when $x < a$ ) . . . . .	$\frac{Pb^2x^2}{6EI/l^3}(3al - 3ax - bx)$